AN EXPERIMENTAL STUDY OF THE EFFECT OF THE AXIAL HEAT FLUX DISTRIBUTION ON THE DRYOUT CONDITIONS IN A 3650-mm LONG ANNULUS

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Abstract---Dryout measurements have been carried out for flow of boiling water in a 3650-mm long annulus. Eight different axial heat flux profiles, typical for BWRs, as well as one with uniform heat flux have been studied. It was found that for the present geometry the effect of the axial heat flux profile upon the total dryout power was rather small or in the range of ± 6 per cent.

1. INTRODUCTION

The basis for dryout predictions in boiling water reactors is full scale rod bundle experiments. However, since such tests are expensive, and available analytical methods still need a strong empirical support, the influence of various parameters has to be established as far as possible by simpler experiments. For two reasons, this is very desirable in the case of the axial heat flux distribution. First, there are large variations in the axial heat flux distribution for BWR fuel assemblies as a function of both core position and time. However, at present those variations are being reduced considerably by an advanced use of burnable absorbers. Secondly, a new set of heater rods is needed each time a new axial heat flux distribution is to be investigated in a dryout experiment, which makes such variations very expensive in the case of full scale 8×8 -bundles.

Systematic investigations of the influence of the axial heat flux distribution on dryout are relatively few in the open literature, even for simple geometries. Among the studies, which cover conditions relevant to BWRs, may be mentioned round tube data by Judd *et al.* (1967) and Becker *et al.* (1969), annular data by Judd *et al.* (1967), Janssen & Kervinen (1963) and Little (1970), and rod bundle data by General Electric (1973) and Lee & Bailey (1977). An interesting analytical contribution is given in a recent paper by Lahey & Gonzalez-Santalo (1977), where a unique relation between the F -factor method by Tong (1972) and the critical quality-boiling length method is demonstrated, and a relation between uniform and non-uniform axial distribution data is derived.

The present investigation was carried out a few years ago to supplement the existing ASEA-ATOM dryout data base, ranging from 4- to 64-rod bundles. It was performed in full length annular geometry and included nine different axial heat flux distributions. Although the results of an annulus are not directly applicable to full scale rod bundles we feel that these data may represent a signifiacant contribution on the way to a full understanding of the influence of axial heat flux distribution on dryout.

2. APPARATUS AND TEST SECTION

The measurements were carried out employing the 1-MW loop at the Royal Institute of Technology. This loop was designed for an operating pressure of 250 bar, and all parts of the loop in contact with water were made from stainless steel. A simplified flow diagram for the loop is shown in figure I. Test sections with heated length up to 7200 mm can be studied, and the power is supplied from a direct current generator. The maximum available current is 6000 A and voltages ranging from 0 to 140 V can be supplied.

Before entering the test section the water passes a 150-kW preheater and a filter. The preheater was used for adjusting the inlet water temperature. After the test section the steamwater mixture flows through a condenser, and the water then enters the circulation pump, which has a pressure head of 100 m of water. The major portion of this pressure head was used in the duct system between the pump and the test section inlet, thus providing sufficient throttling for securing stable operation of the loop.

2.1 *Test section*

The annular test section consisted of two concentric stainless steel tubes. The inner and the outer diameters of the annulus were 12.25 and 20.43 mm, giving an annulus gap of 4.09 mm. The heated length of the inner tube was 3650 mm. It should be noticed that the rod diameter and the heated length were chosen to represent the ASEA-ATOM BWR fuel element geometry.

The inner cylinder was kept in its central position by means of seven sets of spacers, which were mounted on the outer cylinder. The distance between the spacers was 510 mm, and each set of spacers consisted of three 2-mm stainless steel pins, 120° apart. The pins were adjusted by means of micrometers, and were insulated from the outer tube by means of Teflon gaskets. This spacer design had negligible effects on the flow conditions in the annulus.

In order to obtain the desired axial heat flux profiles, the wall thickness of the inner electrically heated tube was varied axially, keeping, however, the outer diameter of the heated tube constant.

At the exit the inner rod was extended with a 150-mm long unheated section. This was achieved by soldering a copper rod to the inside wall of the inner tube.

The water was supplied to the test section through two 20 -mm i.d. ducts 180° apart. A similar arrangement was also provided for the exit steam-water mixture.

As mentioned earlier, the objective of the present study was to test eight rods with different

axial heat flux profiles and one rod with uniform heat flux. After the heater tubes had been received from the manufacturer, the axial variations of the electrical resistance were measured, and it was found that along the full length, the profiles differed less than 3 per cent from the desired values. Figures 2 and 3 show the relative power vs length for the eight non-uniform rods.

2.2 *Dryout detection*

In order to protect the test section from destruction, dryout detectors were used to switch off the power supply when excessive temperature excursions occured on the heated rod. Since it may be possible for dryout to occur at any position downstream of the heat flux peak, the full length of the heated rod downstream of the peak had to be protected. To be able to determine the approximate dryout position, 6 dryout detectors of the Wheatstone bridge type were employed. Three of the detectors used the 2-mm dia. spacer pins as electrodes. However, the 510-mm distance between the spacers was considered too large for a satisfactory determination of the dryout location. Therefore three additional dryout detectors were attached to the heated rod by welding the leads to the inner wall. The leads were taken to the exterior through a hole in the copper bar at the upstream end of the heated tube. Figures 4 and 5 show the locations of the dryout detectors for the eight non-uniformly heated rods.

3. METHOD OF TESTING AND RANGES OF VARIABLES

During the course of a particular run the pressure, the mass velocity and the inlet water temperature were kept constant at prescribed values, while the power supplied to the test section, was gradually increased by small steps until dryout occurred. Just before dryout these steps amounted to less than 1 per cent of the power, and one set of data was taken after each increase by means of a Schlumberger data collecting system. The last set of data obtained before the dryout detectors reacted, was used to evaluate the dryout conditions.

Figure. 2. Axial heat flux profiles.

Figure. 3. Axial heat flux profiles.

In all, 221 runs were obtained at the following conditions:

Figure 5. Locations of dryout detectors.

With regard to the mass velocity the following values were originally selected

 $G = 250, 500, 750, 1000, 1250, 1500, 2000$ and 2500 kg/m^2 s.

In order to verify the reproducibility of the measurements, the experiment started with the low mass velocity and 10°C sub-cooling. After this test series was finished, the 40°C sub-cooling tests were carried out, and finally 10°C sub-cooling measurements were obtained starting at the highest mass velocity and taking data at the following mass velocities.

 $G = 375, 625, 875, 1125, 1375, 1750$ and 2250 kg/m²s.

It was found that the two series of 10°C sub-cooling tests were in perfect agreement with one another, proving the reproducibility of the measurements.

Since the investigation, which covered nine test sections, lasted for more than 3 months, the reproducibility of the equuipment was also checked by running the uniform heat flux test section at two occasions. The research program started with this test section, and after running the eight axial profiles, the uniform test section was again employed to complete the investigation. In the figures the uniform heat flux measurements are referred to as profile 0, and in table I the labels 0 and 9 are used for the initial and the final uniform heat flux measurements. The dryout powers measured at the two occasions differed less than 1 per cent. Hence one may conclude that during the whole investigation the apparatus yielded reproducible dryout results.

4. EXPERIMENTAL RESULTS AND DISCUSSIONS

The experimental results are given in table 1, and in figures 6-13 the data are reproduced in plots of the total dryout power, Q , vs the mass velocity, G . The total dryout power was chosen as the most representative parameter because of the difficulty of determining the exact heat flux at the dryout position. This difficulty was introduced by the dryout detection system, which generally gave the dryout location with a resolution of about 100-200 mm. It should be noticed, however, that for the test section with uniform heat flux dryout always occured at the exit. Also for the test sections 3, 4, 5, 6, 7 and 8 dryout probably occurred at the exit, except for a few runs for test sections 5 and 6 at $G > 2000$ kg/m²s. Exit dryout is indicated by the fact that only the downstream dryout detector reacted, demonstrating the presence of dryout in the down-

Table 1 (Contd.)

Figure 6. Measured dryout conditions.

stream section of the heated rod. For all of the test sections this segment was 150-400 mm long as shown in figures 4 and 5. The dryout locations for the different test sections and runs are found in table 1, where the given coordinate refers to the middle of the section where dryout was observed.

Four methods have previously been used for presenting experimental results for critical heat flux in channels with variable axial heat flux. Little (1970) and Becker (1969), for example, recommended the local hypothesis, while Lee & Obertelli (1963) used the total power hypothesis. Bertoletti et al. (1965) suggested the use of the boiling length hypothesis and Tong et al. (1965) developed the F-factor concept, which takes into account the details of the axial heat flux distribution. It is not the purpose of the present paper to discuss the mentioned hypothesis. This has been done elsewhere, e.g. Lahey & Moody (1975).

Figure 7. Measured dryout conditions.

Figure 9. Measured dryout conditions.

Figure 13. Measured dryout conditions.

Because the present experimental results appeared to be in satisfactory agreement with the total power hypothesis, this rather simple method was adopted for the following presentation of the data. Each of the figures 6–13 shows the experimental results for one axial profile in a plot of the total dryout power vs the mass velocity. To obtain a direct comparison with the uniformly heated test section, the diagrams also include the uniform heat flux results.

The experimental results for all the nine axial profiles are summarized in figure 14.

One observes that for the investigated annular geometry the effect of the axial heat flux distribution is relatively small. This result is supported by the 16-rod bundle measurements reported by General Electric (1973).

Except for a few runs at $G = 2250$ and 2500 kg/m^2 for profile 4, the effect of the flux profile upon the total dryout power is within ± 6 per cent of the results obtained for uniform heating. Although the deviations between the different flux profiles are rather small a few important observations will be mentioned.

The dryout power for the profiles $1, 2, 3, 5, 7$, and 8 is in general low in comparison with the results for uniform heat flux, while the results for profiles 4 and 6 are high compared with uniform heat flux. The higher dryout power for profiles 4 and 6 is reasonable since these profiles have maximum heat flux in the upstream half of the annulus, while the other profiles have the

power peak in the downstream part of the test section or very close to the middle. Considering profile 8, which has two peaks, the data deviates only a few per cent from the uniform flux results.

Figure 15 shows a comparison between profile 1, which has a narrow downstream peak, and profile 4, which has a somewhat similar upstream peak. In the mass velocity range between 750 and 1500 kg/m^2 s, which is of special interest for BWRs the difference in dryout power for the two cases is roughly 8 per cent.

A comparison of profile 6, which has a wide downstream peak and profile 7, which has a wide upstream peak, gives approximately the same difference in dryout power.

Differences in dryout power in the same range between inlet peaked and outlet peaked profiles are indicated by General Electric (1973) for 16-rod bundles, while round tube data by Judd *et aL* (1967) show significantly larger differences. It may be mentioned that the latter data were obtained with 72 in. tubes, which had axial peaking factors close to 2. This would perhaps explain the larger differences in dryout power.

Figure 16 shows a comparison between profile 1 with a narrow downstream peak and profile 7 with a wide downstream peak. The two profiles agree with one another within 2 per cent for mass velocities between 250 and 2000 kg/m²s. Also the profiles 2,3 and 5, which have approximately centered peaks but of different width, and profiles 4 and 6, that have a narrow respectively a wide upstream peak, indicate that the influence of peak width is small.

The dryout power for the uniform profile is significantly lower than for the upstream peaked profiles 4 and 6 and somewhat higher than the rest of the profiles. This is in contradiction to the 16-rod bundle tests by General Electric (1973), where the uniform profile had the highest dryout power of all the tested axial profiles, including one inlet peaked profile.

Lee & Bailey (1977) have compared 36-rod bundle data for uniform and chopped cosine axial profiles. At lower mass velocities there were no significant difference in the dryout power. For mass velocities above $2000 \text{ kg/m}^2\text{s}$, however, the cosine profile was about 6 per cent high. This result is in quite good agreement with our observations, provided profile 5 is considered to be equivalent to a cosine profile.

5. CONCLUSIONS

The most important result of the present investigation is the observation that for the present annular geometry the total dryout power is only to a relatively small extent affected by the axial heat flux profile.

However, a comparison with other investigations indicates that the effect of the axial profile

Figure 15. Measured dryout conditions.

Figure 16. Measured dryout conditions.

is not unique. For rod bundles the situation is quite complex as cross-sectional geometry, spacer distance, spacer design and radial power distribution are parameters, which affect the influence on dryout of the axial heat flux distribution.

As known from rod bundle tests, dryout occurs predominantly just upstream of a spacer in the downstream part of the bundle. In the present tests the pin spacers obviously had negligible influence on the dryout location as with few exceptions dryout always occurred close to the downstream end of the rod. This fact may have contributed to the relatively small influence of the axial power profile. The conclusions obtained from the present study must therefore be applied with caution, when large rod bundles are considered.

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